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# Techno-economic optimization and analysis of a high latitude solar district heating system with seasonal storage, considering different community sizes

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# 6 Abstract

5

A solar community meets a significant amount of its energy demand through solar energy. In a high latitude country like Finland, the seasonal mismatch of solar availability makes it very difficult to achieve high renewable energy fractions without seasonal storage. In this study, a solar community located in Finland was optimized with respect to energy demand and life cycle cost. To gain better understanding of both technical and economical scaling effects, the optimization was done separately for four cases with 50, 100, 200 and 500 buildings.

The study was performed for Finnish conditions using dynamic TRNSYS simulations and optimized with a genetic algorithm, using the MOBO optimization tool. The modeled energy system had solar thermal collectors and solar electric panels for energy generation, two centralized short-term storage tanks and a seasonal borehole thermal energy storage system (BTES) for energy storage, and a ground source heat pump for additional heat generation.

The larger communities provided noticeable cost-benefits when aiming for high performance. Larger seasonal storages allowed more direct utilization of seasonally stored heat, lowering the need for the heat pump and reducing electricity demand. Comparing the best and worst performing optimal energy system, annual demand for heating electricity was reduced by 80%. Renewable energy fractions close to 90% for heating were possible for all community sizes, but the large communities could obtain them with about

<sup>23</sup> 20% lower costs.

24 Keywords: Solar community, Simulation-based optimization, Energy system scaling, Seasonal storage,

<sup>25</sup> Solar district heating, Solar assisted heat pump

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### 26 Nomenclature

27

Symbol / Acronym	Unit	Explanation
BTES		Borehole thermal energy storage
DLSC		Drake Landing Solar Community
DHW		Domestic hot water
HP		Heat pump
PV		Photovoltaics
SH		Space heating
ST		Solar thermal
A <sub>floor</sub>	$m^2$	Heated floor area
A <sub>ST</sub>	$m^2$	Solar thermal area
h <sub>ratio</sub>	m/m	BTES height vs. width ratio
LCC	$\in/m^2$	Life cycle cost over 25 years
N <sub>boreholes</sub>	-	Number of boreholes in BTES
N <sub>series</sub>	-	Number of boreholes connected in series
REF <sub>heat</sub>	-	Renewable energy fraction of heating
REF <sub>total</sub>	-	Renewable energy fraction of total electricity
SF	-	Solar fraction
SPF	kWh/kWh	Seasonal performance factor of heating
V <sub>BTES</sub>	$\mathrm{m}^3$	Seasonal storage volume
$\alpha_{ m tilt}$	0	Tilt angle of solar collectors
hoboreholes	$1/m^2$	Area density of boreholes in BTES

#### 28 1. Introduction

The heating of buildings is a large part of the total European energy demand, especially so in the Nordic 29 countries. For example, in Finland, 87% of energy is consumed by heating (Statistics Finland, 2014). Pro-30 ducing heating through emissions-free renewable energy systems would lower its environmental impact. 31 Such systems might be based on biomass or hybrid solar heating (Modi et al., 2017). Solar energy is a 32 widely available energy source, but suffers from both diurnal and seasonal variation. The diurnal vari-33 ation is a significant problem for solar electric systems, because of the hourly mismatch between energy 34 generation and demand and the high cost of electricity storage. However, thermal energy storage in hot 35 water tanks is a very mature technology and mostly removes the hourly mismatch in heating applications. 36 It can even partly solve the hourly solar electricity mismatch problem (Hirvonen et al., 2016). Unfortu-37 nately, home-scale hot water tanks are of little use in solving the problem of seasonal mismatch, where the heating energy demand is the highest exactly when the solar generation is the lowest, during winter (Figure 4). This problem is especially difficult in high latitude countries, because the relative difference 40 between summer and winter solar energy availability increases the further we move from the equator. 41

The problem of seasonal variation can be solved through seasonal thermal energy storage (Xu et al., 2014). Using seasonal storage, energy can be stored in the peak months to be used during times of high energy demand. While technologies are being developed for chemical and latent heat storage, existing seasonal storage systems mostly utilize sensible heat storage, based on changing the temperature of a high heat capacity material. The basic storage types in this group are hot water tank thermal energy storage (TTES), aquifer thermal energy storage (ATES), water pit thermal energy storage (PTES) and borehole thermal energy storage (BTES).

Seasonal thermal energy storage is often utilized in solar communities, where the goal is to meet a
 significant part of the heating demand by solar energy, that is, to achieve a high solar fraction. The history of
 solar communities began in the 1970s energy crisis (Reuss, 2015). Many such communities have been built
 in Europe in the 1980s, 1990s and 2000s, mostly in Denmark (Heller, 2000), Germany (Schmidt et al., 2004)
 and Sweden (Lundh and Dalenbäck, 2008). Some of the projects store energy into the ground (BTES), but

water-based storage with tanks have also been studied (Tulus et al., 2016). While current efforts in Denmark
 are towards large solar district heating systems based on water pit storage (Ramboll, 2015), existing solar
 communities are of many different sizes, as shown in Figure 1a.

The German Neckarsulm community consists of 200 apartments and a shopping center, school and 57 gym, built in 1997 (Nussbicker et al., 2004). It has a 63 000 m<sup>3</sup> BTES system with a gas boiler and heat 58 pump for backup. The Crailsheim community of similar scale was built in 2007 and has a 37 500 m<sup>3</sup> BTES 59 storage that serves 260 apartments, a school and a gym (Bauer et al., 2010). Backup heating was handled by 60 district heat and a heat pump. The seasonal storage was smaller in Crailsheim than Neckarsulm, but the 61 amount of solar collectors was larger, 7500 m<sup>2</sup> compared to 5670 m<sup>2</sup>. The Attenkirchen solar community is 62 much smaller, serving only 30 homes (Reuss et al., 2006). This system utilizes an underground water tank, 63 surrounded by a 10 500 m<sup>3</sup> BTES system. Similar design was used in the only Finnish solar community 64 trial in Kerava (Lund, 1984), though the system was later dismantled and replaced by conventional district 65 heating. 66

Perhaps the most famous solar community is the Drake Landing Solar Community (DLSC) in Canada, 67 which started operation in 2008 (Sibbitt et al., 2011). It utilizes 2300  $m^2$  of solar collectors, two water-based 68 buffer storage tanks and 34 000 m<sup>3</sup> BTES system to supply heating to 52 houses. A system of similar scale 69 is the Swedish Anneberg solar community, with a 60 000<sup>3</sup> BTES volume and a 2400 m<sup>2</sup>solar thermal area. 70 The DLSC system has been able to meet 98% of space heating demand through solar energy, while the 71 Anneberg system supplies about 60% of combined space heating and domestic hot water (DHW) demand 72 (Zhu, 2014). On the other extreme of solar communities is the Braedstrup solar district heating system in 73 Denmark (SDH EU, 2012). It was built in 2007 and extended in 2012 to have a 19 000 m<sup>3</sup> BTES volume 74 with an 18 600  $m^2$  solar thermal area. The system is backed up by an electric boiler and a heat pump and 75 supplies heat to 1200 homes. Because of the large heat demand compared to the solar thermal capacity, the 76 solar fraction is only 20%, but this also ensures minimal waste of available solar energy. 77

Every solar community has a different amount of buildings, different sizes of short-term and long-term 78 energy storage as well as different solar collector areas, different auxiliary heating systems and different 79 environmental conditions. Thus, even when the communities report different solar fractions, it is hard 80 to tell what is the main reason for the performance. Figure 1b shows the solar fractions achieved by solar 81 communities, arranged according to their ratio of energy storage to solar capacity and ratio of solar capacity 82 to heated space. All of these systems utilize BTES for their seasonal storage needs. It seems that the highest 83 solar fractions have been achieved by systems with more solar thermal capacity per heated area and more 84 storage capacity per solar thermal capacity. The opposite also holds true for the smallest solar fraction. 85

<sup>86</sup> However, for most samples the correlation is far from clear, which implies other factors are also important.





(a) Solar fraction of some realized solar communities.

(b) Solar fraction reported with relative system sizes.

This article examines the effect of community size on the techno-economic optimization of a high lat-87 itude solar community in a heating-dominated climate. Specific environmental challenges include high 88 seasonal variability of solar energy and highly conductive ground. Total system optimizations have not been widely reported in the literature. The focus is specifically on Finland, as actual solar community 90 projects have not been materialized there like in other Nordic countries. However, neighbouring countries 92 Sweden, Norway and Estonia share a similar climate which increases the applicability of the results. The work is done by multi-objective optimization with a genetic algorithm, using life cycle cost and energy performance as objectives. Different energy storage and generation configurations are considered, along 94 with efficiency improvements. The aim is to see if community size has an influence on which features are 95 dominant and to increase understanding of system design to help realize actual projects in the near future. 96

## 97 2. Materials and methods

#### 98 2.1. Energy system details and modeling

The solar community under study consisted of a simulated group of  $100 \text{ m}^2$  single-family detached 99 houses, which were connected to a local heating grid. The buildings were assumed to be located in south-100 ern Finland (60 °N) The energy system was modeled in TRNSYS 17. Solar thermal collectors (ST) and 101 solar photovoltaic panels (PV) were located on the roofs of each building. The solar thermal system was 102 connected to a centralized heat storage system, containing two water-based short-term storage tanks (high 103 and low temperature) and a borehole thermal energy storage (BTES) system (Type 557a). The design of the 104 system is shown in Figure 2. The solar collectors were connected to both tanks in parallel, so each could be charged according to current temperature levels. The low temperature tank, which was kept at close to 106 40 °C, was used for supplying space heating (SH) and for preheating domestic hot water (DHW). The high 107 temperature tank, which was kept at above  $60\,^\circ\mathrm{C}$ , was used for boosting the DHW to the minimum tem-108 perature of 55 °C required by the Finnish building code (Finlex, 2007). The solar thermal output was used 109 to keep the hot tank at the required temperature, otherwise the flow was directed to the low temperature 110 tank. If the tank temperature rose 10  $^{\circ}$ C above the setpoint, the energy in the tank was discharged into 111 the seasonal storage until the tank had cooled down enough. Solar thermal collectors were modeled based 112 on Savo-Solar rooftop collectors (Solar Keymark Database, 2013). The flowrate to the solar collectors was controlled to keep the temperature output at  $1^{\circ}$ C higher than the top of the target thermal storage tank. 114 Either tank could be charged, depending on the current storage temperatures. 115

The seasonal storage was a borehole thermal energy storage (BTES), where the boreholes were evenly 116 distributed along the top surface area. Each borehole was fitted with U-tube piping which served as a heat 117 exchanger between the ground and the heat transfer fluid. Several boreholes could be connected in series 118 so that the fluid exiting from one borehole could be pumped into the next one. In charging mode, hot fluid 119 was pumped into the center of the storage and the cooled output flow was directed into the next borehole 120 in series. Thus, a radial temperature distribution could be formed, where the center of the storage had 121 the highest temperature and the edges had the lowest, minimizing heat losses to the surroundings. The 122 flowrate in each borehole loop was set to 400 kg/h, but if during charging the tank temperatures rose too 123 high, the flowrate was doubled to prevent the tank from overheating. 124

When the temperature in the tanks was too low, they were charged from the BTES. In this mode, the cold 125 fluid entered from the cool outer edge and exited from the hot center. Whenever possible, the flow from 126 BTES was used to heat the tanks directly, but if the temperature was not high enough it was instead used 127 as the heat source for heat pumps. The high temperature of the ground circuit significantly increased the 128 COP, as shown in Figure 3. Direct electric heaters were used for backup heating. If the ground was cooled 129 to below  $-5 \,^{\circ}$ C, the heat pump was deactivated and heating was handled by the direct electric heaters. If 130 the source temperature for the heat pump was above 30 °C, the source flow was assumed to be mixed with 131 the cooled return flow, so that the heat pump COP matches the maximum values shown in Figure 3. 132

Cooling and space heating demands in the building were based on TRNSYS simulations with Type 56.
 Different building configurations were simulated independently of the solar community, using different
 options of windows, insulation and heat recovery efficiency. Seven Pareto optimal building configurations



Figure 2: The heating system consisted of solar thermal collectors, two buffer tanks, a borehole seasonal energy storage and a heat pump. Parallel connection allowed solar collectors to charge either tank separately, while series connection treated them as a single tank.



Figure 3: The COP of the heat pump increases until the source temperature reaches 30 °C. Above that point the inlet flow was assumed be mixed with cooled return fluid so that the COP remained constant.

(in terms of cost and energy efficiency) were used as options for the total system optimization, where LCC and external electricity use of the community were similarly minimized. The annual space heating demand with respect to floor area in the buildings varied from 25 kWh<sub>th</sub>/m<sup>2</sup> to 50 kWh<sub>th</sub>/m<sup>2</sup> (Figure 7). Due to the low cooling needs of the community, no cooling system was included.

The DHW flow profile was based on IEA data (Jordan and Vajen, 2001). Several copies of the same 140 profile were shifted by 0 to 5 weeks and 0 to 2 hours to provide an aggregation effect where the weekdays 141 remain the same, but slight differences to hourly scheduling appear. The basic annual DHW demand in the 142 buildings was 35 kWh<sub>th</sub>/m<sup>2</sup>, but to guarantee quick access to hot water the simulation included constant 143 recirculation of hot water, even when there was no actual demand, which increased the effective use to 144  $42 \text{ kWh}_{th}/\text{m}^2$ . Additionally, the demand profile for lighting and appliances was a normalized aggregate 145 of measured demand from 50 buildings in the Helsinki region, provided by the Fortum company (Fortum 146 Oy, 2015). For privacy reasons, no other information about the houses was provided and the aggregated 147 profile was thus normalized to 40 kWhel/m<sup>2</sup> annual electricity demand, according to measurements of a 148 low-consumption house where size data was available. 149

Figure 4 shows the monthly heating demand per heated floor area and the solar insolation for a surface sloped at 40°. DHW demand remains roughly constant through the year, while SH demand goes to zero during the summer. The figure also shows the difference in SH demand in the best and worst quality houses, with the space heating demand for all seven building configurations lying between the minimum and maximum values. The seasonal mismatch between demand and generation is evident, along with the large difference between the minimum and maximum solar generation.

> 14 200 180 12 kWh/m<sup>2</sup> 160 Heating demand (kWh/m<sup>2</sup>) 10 140 adiation 120 8 100 6 80 'n C V 60 4 ncident 40 2 20 0 0 2 5 6 7 8 9 1 3 4 10 11 12 Month DHW SH minimum SH maximum Solar

Figure 4: Monthly heating energy demand as thermal energy. Space heating demand is split into two parts: the minimum corresponds to the most efficient building configuration 7, while the maximum corresponds to the least efficient building configuration 1 (Figure 7). Solar insolation is given for south-facing collectors tilted at a 40 ° angle.

The weather data used for the simulation was the Helsinki Test Reference Year 2012 (Kalamees et al., 2012), with annual horizontal solar insolation of 975 kWh/m<sup>2</sup>. The BTES was located on a rocky Finnish ground, with an average thermal conductivity of 3.50 W/mK. The volumetric heat capacity was 2240 kJ/m<sup>3</sup>K (Flynn and Sirén, 2015). Because the BTES system required several years to heat up and achieve optimal performance, it was not enough to run the simulation for just one year, but due to long simulation time, 25 year simulation was also not considered feasible. As a compromise, the system was simulated for 4 years and the fourth year was used to estimate the cost and performance for all the remaining years.

#### 163 2.2. Performance indicators

The performance of a solar community can be evaluated by examining its final demand for imported 16 off-site energy. However, some other indicator can also be useful for comparing case studies. Solar fraction 165 (SF) is a common indicator that describes the fraction of energy demand met by solar energy. The solar 166 fraction is also known as the renewable energy fraction or the on-site energy fraction (Cao et al., 2013). 167 In this study, the solar energy system is supported by a ground-source heat pump, which means that 168 energy can be taken out of the storage even if it has not been charged with active solar systems. Thus, 169 to avoid confusion, this study uses the renewable energy fraction (REF) as a performance indicator. REF 170 was defined separately for heating and total electricity demand. Due to utilizing both short-term and long-171 term thermal storage, as well as heat pumps, the total solar energy generation couldn't be used for the 172 calculation directly, because there were significant thermal losses and energy conversion from electricity 173 to heat. Instead, REF was calculated indirectly by determining the electricity consumption of all heating-174 related systems (HP, pumps, backup heating) and assuming all remaining energy was generated by solar 175 energy. For heating, REF was defined using the ratio of electricity needed to run the heating system vs. the 176 total heat demand met by the system 177

$$REF_{\text{heat}} = 1 - \frac{E_{\text{heating}}}{E_{\text{heat,dem}}} = 1 - \frac{E_{\text{pumps}} + E_{\text{HP}} + E_{\text{backup}}}{E_{\text{SH}} + E_{\text{DHW}}},\tag{1}$$

where  $E_{\text{heating}}$  is the electricity needed to run the heating system,  $E_{\text{heat,dem}}$  is the total heating energy used in the buildings,  $E_{\text{pumps}}$ ,  $E_{\text{HP}}$  and  $E_{\text{backup}}$  are the electricity use of the pumps, heat pump and backup heating, respectively, and  $E_{\text{SH}}$  and  $E_{\text{DHW}}$  are the actual amount of heat used to provide space heating and domestic hot water.

Equation 1 indirectly takes into account both the direct-use and ground-stored solar energy, as their inclusion lowers the need for direct electric backup heating and heat pump compressor operation (which is denoted by  $E_{\rm HP}$ ). Solar electricity was not considered in the calculation of REF<sub>heat</sub>.

Since grid electricity was the only external energy source, the REF for the whole system was defined using the ratio of annually imported electricity vs. the total electricity demand

$$REF_{\text{total}} = 1 - \frac{E_{\text{elec,import}}}{E_{\text{elec,dem}}} = 1 - \frac{E_{\text{heating}} + E_{\text{appliances}} - E_{\text{PV-self-consumption}}}{E_{\text{heating}} + E_{\text{appliances}}},$$
(2)

where  $E_{\text{elec,import}}$  is the electricity imported from the grid,  $E_{\text{elec,dem}}$  is the total electricity use by the buildings,  $E_{\text{appliances}}$  is the energy demand of lights and other non-heating appliances and  $E_{\text{self-consumption}}$  is the amount of locally generated solar electricity that is used on-site (not exported as excess). Excess solar energy is sold to the grid for small profit, but it does not compensate for demand in other time periods.

The efficiency of a seasonal energy storage system can be defined simply as the ratio of annual energy taken out of the BTES vs. solar energy injected into the storage (Flynn and Sirén, 2015)

$$\eta_{BTES} = \frac{E_{discharge}}{E_{charge}}.$$
(3)

This efficiency measure is dependent on both the storage and energy generation system performance and can sometimes give unexpected results. For example, if there is not enough heat injected into the system, the annual discharging may exceed charging, giving an efficiency greater than 1. When this happens, the storage is either recharged naturally from the surrounding ground or the BTES is permanently cooled, eventually making it unusable for geothermal energy applications.

## 198 2.3. Economic parameters

The life cycle cost (LCC) was formed out of the initial investment cost and the operation cost (cost of imported electricity) during 25 years. Discounting was done with a real interest rate of 3% (EU-Commission, 2012). The hourly electricity price was the sum of the Nord Pool spot price (Nord Pool, 2016), the distribution price of the local network operator and the Finnish electricity tax (Caruna Oy, 2016). The average prices (including VAT) were 5.5, 4.0 and 2.8 c€/kWh, respectively. The total cost for buying electricity was thus approximately 12 c€/kWh. The value of self-consumed PV electricity was equal to the total buying price. However, when selling excess electricity to the grid, the value was equal to only the spot price. Hourly spot price profiles from the years 2010 to 2014 were used cyclically, so that each profile repeated every five years. Due to the reversing price trend in the Nordic electricity market, the average price increase during the past decade has been low and even negative (Nord Pool, 2016). Thus, a conservative electricity price escalation rate of 1% was used in this study.

A constant unit price was used for some energy system components (Table 1). For other components, 210 the price was assumed to go down with larger amounts. These include the solar thermal collectors (Figure 211 5), PV panels (Figure 5) and tanks (Figure 6). Lowering relative price results from economies of scale, as 212 fixed costs may be independent of system capacity and production efficiency improves by making larger 213 production runs and through worker learning. Companies can also profit, if they can sell more even with 214 a slightly lower price. The price for improving the energy performance of buildings was given as the 215 difference against the worst performing building configuration (Figure 7). Costs of building efficiency 216 components are reported in Table 2 and the optimal building configurations are shown in Table 3. 217

Tabl	e 1:	Energy	system	components	with	constant	unit prices.
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System component	Price	Unit
HP (Haahtela and Kiiras, 2013) Borehole drilling (Haahtela and Kiiras, 2013) BTES excavation (Rakennuslehti, 2016) BTES insulation (Rautia, 2016)	325 33.5 3.0 88	$\begin{array}{c} { \displaystyle \displaystyle \in / k W } \\ { \displaystyle \displaystyle \displaystyle \in / m } \\ { \displaystyle \displaystyle \displaystyle \displaystyle \displaystyle \displaystyle \displaystyle \in / m^3 } \\ { \displaystyle \displaystyle$

Table 2: Cost of building efficiency components (Hamdy et al., 2013; Haahtela and Kiiras, 2013).

Component	Details	Price	Unit
Wall insulation	Mineral wool	62.7	€/m <sup>3</sup>
Roof insulation	Blow-in wool	36.2	$\in/\mathrm{m}^3$
Floor insulation	Polyurethane	111.4	$\in/\mathrm{m}^3$
	U-value $0.98 \text{ W/m}^2\text{K}$	195	
Windows	U-value $0.8 \mathrm{W/m^2K}$	221	$\in/\mathrm{m}^2$
	U-value $0.6 \mathrm{W/m^2K}$	270	
	Cross-flow HX, $\eta = 60\%$	3533	
Ventilation heat recovery efficiency	Counter-flow HX, $\eta = 70\%$	3835	€/house
	Regenerative HX, $\eta=80\%$	4138	

Insulation and window U-values										
Configuration	SH demand	Wall	Floor	Roof	Windows	Heat recovery efficiency				
	$(kWh/m^2)$	$(W/m^2)$	$(W/m^2)$	$(W/m^2)$	$(W/m^2)$	(%)				
1	50.2	0.17	0.17	0.09	0.98	60				
2	43.0	0.17	0.17	0.09	0.98	70				
3	37.3	0.17	0.17	0.09	0.98	80				
4	34.2	0.13	0.17	0.09	0.98	80				
5	31.2	0.13	0.17	0.07	0.98	80				
6	28.9	0.10	0.17	0.07	0.98	80				
7	25.1	0.13	0.17	0.07	0.60	80				

Table 3: Optimal building configurations.



Figure 5: Cost of rooftop solar thermal (Mauthner, 2016) and solar electric (Ahola, 2015) systems. Prices are given as  $\in/kW_{th}$  and  $\in/kW_{el}$ , respectively. Solar thermal power estimated at 10 K temperature difference.



Figure 6: Cost of short-term thermal storage tanks (Mauthner, 2016).



Figure 7: Additional cost (per floor area) of building energy efficiency improvements relative to the lowest performing building. These include different levels of insulation, window quality and heat recovery efficiency as shown in Tables 2 and 3.

#### 218 2.4. Optimization problem

The goal of the optimization was to minimize both imported electricity and life cycle cost (LCC). The problem was defined as

$$\min \{ f_1(x), f_2(x) \}$$
s.t.  $g(x) = A_{ST} + A_{PV} \le 60 \text{ m}^2$ 

$$lb_i \le x_i \le ub_i, \ i \in \{1, ..., 14\},$$

$$(4)$$

where  $f_1$  is the amount of annually imported electricity,  $f_2$  is the life cycle cost, g is the limit on roof space and  $lb_i$  and  $ub_i$  are the lower and upper bounds for all decision variables. The decision variables are introduced in Table 4. The life cycle cost was the sum of initial investment cost and the discounted operation cost over 25 years.

Decision variable	Lower bound	Upper bound	Unit	Variable type	Description
$\begin{array}{c} A_{ST} \\ V_{hot} \\ V_{warm} \\ A_{PV} \\ h_{ratio} \\ V_{BTES} \\ d_{insulation,BTES} \\ \alpha_{tilt} \\ E_{building} \\ N_{HP} \end{array}$	2 0.5 0.5 0 0.25 100 0 20 25 1	40 5.0 5.0 60 5 1000 2 80 50 25	m <sup>2</sup> /building m <sup>3</sup> /building m <sup>2</sup> /building - m <sup>3</sup> /building m ° kWh/m <sup>2</sup> /a	Continuous Continuous Continuous Continuous Continuous Continuous Continuous Discrete Discrete	Solar thermal collector area on each roof Hot buffer tank size Warm buffer tank size Solar electric panel area on each roof BTES height vs. width ratio Volume of seasonal borehole storage Thickness of BTES top insulation Tilt angle of solar collectors and PV panels Space heating demand in buildings Number of heat pumps (60 kW <sub>th</sub> )
$ ho$ boreholes $\mathrm{N}_{\mathrm{series}}$	0.05 1	0.25 9	boreholes/m <sup>2</sup> -	Continuous Discrete	Borehole density Number of boreholes in series

Four different community sizes were used: 50, 100, 200 and 500 buildings. A separate optimization was performed for each community size. Optimization was performed with the MOBO optimization tool (Palonen et al., 2013), using a Pareto archive genetic algorithm (Hamdy et al., 2012). Population size was 24, crossover probability 0.9 and mutation probability 0.07. Additional calculations during optimization were performed with MATLAB.

### 230 3. Results

#### 231 3.1. Optimization results

The results of the optimization for all community sizes are shown in Figure 8. When increasing the 232 community size from 50 to 200 buildings, the same performance was obtained with lower cost. Further 233 size increase to 500 buildings provided no additional cost benefit. It is clear that lower unit costs of larger 234 systems should provide some economical benefits, but are there other factors in play as well? Are there 235 some pure performance benefits or specific features that follow from larger system size? All the following 236 analysis is based on the near-optimal solutions shown in Figure 8. Many figures show the results with 237 respect to imported electricity, which means the total electricity demand of heating and appliances after 238 239 self-consumption of PV power has been subtracted. Excess solar electricity generation was sold for profit, but did not affect the net energy balance. 240

Table 5 shows the features and performance of some Pareto optimal solutions for each community size.

<sup>242</sup> They were chosen based on LCC, so that each size category has three cases of similar cost. This helps to



Figure 8: Pareto optimal solutions for each community size, showing discounted life cycle cost during 25 years vs. the consumption of total imported electricity after PV self-consumption has been accounted for. The values are normalized to the total heated floor area of all buildings in the community.

illustrate differences in performance and cost distribution. We can note that the short-term storage capacity increases when aiming for high performance. On the other hand, large seasonal storage volumes were connected with the lowest performance. When the community size increases, a high performance seems to require less boreholes per building. There were 7 different options for the building energy performance (1 having the worst performance and 7 the best) and most optimal configurations used building configuration 7. This indicates that minimizing energy demand has a higher priority than investing in energy generation and storage systems.

Figure 9 shows the BTES efficiency with respect to the total purchased electricity consumption. On the 250 high performance side (energy consumption between 30 and 35 kWh/ $m^2$ ), the larger communities had 251 better efficiencies than the smaller ones as the efficiency increased from 40% in the 50 building case, to 252 55% in the 100 building case and up to 70% in the 500 building case. This matched with expectations, but 253 for much of the samples there was no clear difference in BTES efficiency that could be attributed to the 254 community size. For all sizes, a clear trend is visible, as storage efficiency decreased while total system 255 performance improved. This can be attributed to the increased storage temperatures resulting from larger 256 ST capacities. 257

BTES efficiency as defined in Equation 3 can be misleading, as evidenced by the points where the ef-258 ficiency is greater than 1. These situations occur when the solar heating system is undersized and heat 259 pumps drain so much energy from the ground that it will cool down below the temperature of the sur-260 rounding ground. When this happens, the BTES is naturally regenerated like a conventional GSHP system 261 and the BTES efficiency increases. Thus, BTES efficiency is highest when solar energy injection is minimal. 262 To enhance readability, abnormally high efficiencies that were greater than 10 are not shown in the figure, 263 even though such cases occurred in the case of 50 buildings when solar thermal capacity was less than 264  $10 \text{ m}^2$ . 265

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Buildings			50			100			200			500	
Category		А	В	С	A	В	С	A	В	С	A	В	С
Electricity use	kWh/m <sup>2</sup>	45.6	36.2	31.7	47.1	34.1	30.9	45.8	32.3	28.9	43.0	32.7	28.3
LCC	€/m <sup>2</sup>	269	329	399	269	330	399	273	331	403	270	332	393
SPF	kWh/kWh	3.1	5.3	8.1	2.9	5.5	8.3	3.0	6.8	11.3	3.3	7.3	13.2
REF <sub>heat</sub>	-	0.68	0.81	0.88	0.65	0.82	0.88	0.67	0.85	0.91	0.70	0.86	0.92
REF <sub>total</sub>	-	0.28	0.33	0.35	0.27	0.36	0.37	0.30	0.36	0.38	0.30	0.35	0.38
ST area	m <sup>2</sup> /house	6.3	19.6	32.0	5.3	20.8	26.4	13.7	22.8	32.9	11.1	25.3	28.7
Hot tank volume	m <sup>3</sup> /house	0.6	1.7	1.0	0.7	1.7	4.2	0.7	1.8	3.0	0.5	0.6	3.5
Warm tank volume	m <sup>3</sup> /house	1.0	1.9	2.6	0.8	1.8	3.2	0.6	2.3	2.9	0.8	4.4	4.2
PV capacity	kW/house	3.5	4.0	4.0	3.3	5.3	5.0	4.7	5.3	4.2	4.2	4.9	4.4
BTES height-to-width	m /m	5.00	1.32	0.73	3.76	1.19	0.97	1.32	0.89	1.74	3.41	1.04	0.68
BTES volume	m <sup>3</sup> /house	859	286	250	1401	196	247	229	200	411	553	165	359
Borehole density	$1/m^2$	0.15	0.16	0.16	0.12	0.21	0.20	0.12	0.13	0.21	0.23	0.17	0.15
<b>BTES</b> Insulation	m	0.03	0.10	0.55	0.06	0.04	1.10	0.08	0.21	0.65	0.06	1.19	0.10
Solar tilt	0	53	53	52	53	43	55	43	48	43	50	49	49
Building quality	-	7	7	7	7	7	7	5	7	7	7	5	7
Number of heat pumps	-	2	1	1	3	2	2	5	4	5	14	12	12
BTES width	m	22	24	28	36	28	32	35	38	39	47	47	69
BTES height	m	111	32	20	136	33	31	47	34	68	160	48	47
Number of boreholes	-	60	72	99	120	124	159	118	146	255	392	291	564
Boreholes in series	-	3	6	9	3	2	3	2	2	3	2	1	3
Boreholes/house	-	1.20	1.44	1.98	1.20	1.24	1.59	0.59	0.73	1.28	0.78	0.58	1.13



Figure 9: Average BTES efficiency over the whole lifetime. BTES efficiency was defined as the ratio of discharged vs. charged energy.

BTES efficiency values were related to the shape of the seasonal storage. Figure 10 shows the height-266 to-width ratio for the optimal cases with all community sizes. With 50, 100 and 500 buildings, the height-267 to-width ratio was high when the system performance was poor. This means that the seasonal storage is 268 more deep than wide. This increases the heat transfer through the sides of the storage. When solar energy 269 injection is low, a cooled down BTES will be charged naturally from the surrounding terrain. Performance 270 improvement was generally tied to a decreasing height-to-width ratio. The BTES can be insulated from the 271 272 top, which benefits wide designs with a low height-to-width ratio. A wide shape also allows the BTES core to obtain higher temperatures. The decreasing height-to-width ratio can also be seen in Figure 11. A wide 273 shape has the added benefit that more boreholes can be installed, which can increase total flowrate. When 274 there is little solar injection, the ground storage needs to be very large, to prevent freezing during heat 275 pump operation, as can be seen in Figure 12. In the low performance scenarios (over  $45 \text{ kWh/m}^2$  electricity 276 demand), a large community seems to need relatively less storage volume to obtain similar performance as 277 a small community. In the case of 200 buildings, all optimal solutions had height-to-width ratios close to 1, 278 even in the low performance cases. Clearly, BTES design is not the only factor that determines total system 279 efficiency. The difference of the 200 building scenario can be explained by its higher solar thermal capacity 280 (Figure 13). Even in the low performance scenarios ( $45-50 \text{ kWh/m}^2$ ) there was enough solar injection to 281 allow the wide and small storage to be feasible. In the 50 and 100 building communities the lack of solar 282 energy had to be compensated by a large storage with significant passive regeneration. In the 500 building 283 community, both the average BTES size and solar capacity were between these two extremes. Some of 284 the differences between the size categories can be explained by the random nature of genetic algorithms. 285 Solutions generated by GA depend on both previous solutions and random changes and are not guaranteed 286 to be optimal, though in practice they are usually nearly optimal. 28



Figure 10: Height-to-width ratio of BTES for the optimal cases.

As the size of the community was increased, the number of boreholes connected in series in the BTES 288 decreased. The amount of seriality in the optimal cases is shown in Figure 14. It was expected that having 289 more seriality would improve total efficiency by increasing the BTES output temperature and by lowering 290 thermal losses through stronger radial temperature distribution. Indeed, this was the case in the 50 build-291 ing scenario, as high performance was obtained using 4 to 9 boreholes in series. However, in the larger 292 communities the number of series-connected boreholes was lower. Since the flowrate within each borehole 293 loop was kept constant, parallel borehole connections allowed for a higher total flowrate, which increased 294 the power transfer to and from the BTES. Despite the higher losses in the seasonal storage, this proved 295 beneficial for the cost of operation in cases with 100, 200 and 500 buildings, as more of the heating demand 296



Figure 11: Height-to-width ratio of BTES for the optimal cases, averaged based on total purchased electricity consumption.



Figure 12: Average size of BTES in each performance category.



Figure 13: Average size of solar collector arrays in each performance category.

<sup>297</sup> could be met by direct utilization of seasonally stored heat, without heat pump operation. Having many

<sup>298</sup> boreholes connected in series increased the output temperature from the BTES. While this made it easier

to directly utilize the stored heat, it also caused the output temperatures to be unnecessarily high, which

could induce mixing losses when the low temperature tank was charged with unnecessarily hot fluids. Additionally, the heat pump gained no benefit from temperatures higher than 30 °C. It seems that strong

Additionally, the heat pump gained no benefit from seriality is only important for small BTES sizes.



Figure 14: Number of boreholes in series for the optimal cases.

One benefit of larger community size was that the relative amount of boreholes decreased. Figure 15 shows the number of boreholes per house. The amount increased as performance improved, but the larger communities had lower average values. Drilling can be a significant cost, so the lower amount is a benefit for large communities. The number of boreholes does not tell the whole truth, however. Figure 16 shows the combined length of all boreholes relative to the number of buildings. Here we see an opposite trend that the total borehole length decreased while performance improved. This implies that high performance
 is achieved by having many shallow boreholes on a wide area.



Figure 15: Average number of boreholes per building in each performance category.



Figure 16: Average total borehole length per building in each performance category.

The fraction of heating provided directly from the BTES without HP is shown in Figure 17. It can be 310 seen that only systems above a certain minimum performance threshold could provide any direct use of 311 the seasonally stored heat. Without enough solar heating capacity, the temperature in the boreholes never 312 rises high enough and thus the heat pump needs to be utilized. The lower temperatures will lower heat 313 losses, while the use of the heat pump will increase the fraction of utilized heat, thus improving BTES 314 efficiency (Figure 9). The maximum bypass fraction for the 50 and 100 building communites were 0.20 and 315 0.15 respectively, while with 200 and 500 buildings it was 0.43. Thus, with a larger community, the need for 316 317 heat pump utilization was lower, but increasing the size from 200 to 500 did not provide additional benefit. All the optimal solutions had varying solar thermal and seasonal storage capacities. Figure 18 shows 318



Figure 17: The fraction of heat provided directly from BTES, bypassing the heat pump.

the renewable energy fraction of heating vs. the ratio of BTES volume to solar collector area (storage-319 to-generation). REF<sub>heat</sub> values below 0.74 were obtained by a wide range of storage-to-generation ratios. 320 However, when the storage-to-generation ratio was below 20, a roughly linear trend could be observed, 321 as a decreasing ratio was correlated with an increasing REF<sub>heat</sub>. The performance values of all community 322 sizes were mixed together and no size effects could be inferred, though in the low REF<sub>heat</sub> cases the 500 323 building community had higher relative storage volumes than the other sizes. A cluster of high renewable 324 energy fractions for all sizes can be seen where the storage-to-generation ratio is between 5 and 10. This 325 ratio could be used as a simple design rule when planning a system with high intended use of solar energy. 326 Ignoring performance values, Figure 19 shows the relative seasonal storage and solar thermal capacities 327 for all the optimal cases. The corresponding values from realized solar communities were also added for 328 comparison. Most realized cases had values similar to those reached in the optimization, except for Braed-329 strup (which was designed for a low solar fraction) and Anneberg (which had the highest solar fraction 330 of the real cases but had no heat pump). When the ST area in the simulated cases was very low, it was 331 compensated for with a large BTES volume. In these regions, each size category clustered in different ST 332 capacity ranges, so they are not directly comparable, but the relative storage volume decreased as gener-333 ation increased. When the ST area exceeded 13 m<sup>2</sup>, all community sizes tended to have storage volumes 334 close to 2  $m^3/m^2$ . The smallest community size had the highest solar thermal capacities, which points to 335 less efficient seasonal storage, as thermal losses need to be compensated with more generation. This only 336 applies to the high performance group, however, as shown in Figure 13. Energy performance was strongly 337 tied to the solar thermal area, but in the low performance regions (energy demand over 40 kWh/m<sup>2</sup>) each 338 size category had different approaches to total performance, as evidenced by varying solar thermal areas. 339 General trends for short-term storage tanks can be seen in Figure 20. All community sizes show a similar 340 trend, where more than 2 m<sup>3</sup> of buffer storage was only needed to keep purchased electricity demand 341 below 35 kWh/ $m^2$ . The standard deviation for the tank sizes was greatest between demand from 30 to 342  $35 \text{ kWh/m}^2$ . In most cases the warm tank was larger than the hot tank. This was to be expected, as 343 most of the solar energy was stored in the low temperature tank. In fact, an even larger difference in 344 tank sizes was expected, because roughly 70% of the thermal energy demand was estimated to be met 345 through the low temperature tank. Thus a size difference as high as 130% could be expected. However, 346 the average size difference was 70%, 1%, 30% and 35% for the community sizes of 50, 100, 200 and 500 347 buildings, respectively. The small difference may indicate some need to adjust temperature set points or 348

<sup>349</sup> charge control algorithms.



Figure 18: Renewable energy fraction of heating vs. storage volume to collector area ratio.



Figure 19: Relative BTES volume and solar thermal area for each community size. Values from realized solar communities (Figure 1b) are shown for comparison.



Figure 20: Average sizes of short-term storage tanks in each performance category.

The total combined area of ST collectors and PV panels was limited to  $60 \text{ m}^2$ . Figure 21 shows the area 350 of each collector type for all community sizes in the optimal cases. In most cases, PV panels occupied 351 the majority of the used space and in all cases the total area approached the upper limit. For 50, 100 and 352 500 buildings, there was a rising trend in ST area after going below 40 kWh/m<sup>2</sup> import need. Changes in 353 PV area were more random in all scenarios. Self-consumption of PV power was not significantly increased 354 above 3 kW capacity (about 20 m<sup>2</sup>), which explains the small differences in area between the best and worst 355 performance. Having a very large PV area was not useful, because the value of exported solar electricity 356 was less than that of self-consumed electricity. On average, the best performance was obtained with a 357 near-even split between the two solar collector types. 358

The LCC was comprised of initial investment costs and lifetime operation costs. Operation cost includes 359 only electricity purchased during the 25 years of operation, but no maintenance. Figure 22 displays the cost 360 distribution for some optimal cases, chosen to have similar total costs (Table 5). In the cheapest category A, 361 the 200 building case had increased heating electricity demand compared to the 50 and 100 building cases. 362 A noticeable difference was a larger investment to ST capacity and a smaller investment to seasonal storage. 363 Investment to PV systems stayed roughly the same for all categories, because self-consumption potential 364 limited the maximum sensible amount of solar electricity, since the selling price of excess electricity was 365 significantly lower than the value of self-consumption. Higher performance of categories B and C was 366 fueled by increased investments to buffer tanks and solar thermal capacity, but also to BTES system in 367 category C. 368

All the previous results have focused on the total electricity consumption, which includes both the 369 appliances and the heating system. However, to better understand the solar energy system's true perfor-370 mance, one should examine the heating energy separately. Figure 23 shows how the heating system used 371 electricity in the optimal cases of the 500 building scenario. The electricity demand of the heating system 372 reduced from over 25 kWh/m<sup>2</sup> in the worst case to 5 kWh/m<sup>2</sup> in the best case, an 80% reduction. In the 373 low performance cases with little solar energy, the heat pump required significant support from the backup 374 heating system. As the solar thermal area increased, more of the heating demand could be directly met by 375 solar energy, but additionally the COP of the heat pump was improved and the use of the HP bypass was 376 increased. 377



Figure 21: The use of roof area by ST collectors and PV panels in the optimal cases of each community size.



Figure 22: The distribution of relative lifetime costs between different components for cases with similar LCC. Also shown is the annual electricity used for heating, as well as the total purchased electricity consumption. The letters A, B and C refer to the community cost category, from cheapest to most expensive, as shown in Table 5. The largest communities can be seen to have somewhat lower electricity consumption in each cost category.



Figure 23: The distribution of electricity used for heating in the optimal solutions of the 500 building scenario.

## 378 4. Discussion

Almost all of the optimal cases included the best-performing building configuration, number 7. This means that reducing heating demand is perhaps the most important step in high latitude solar community design. Saving energy seems to be more cost-effective than adding more generation, especially considering the significant losses entailed in seasonal thermal storage.

The optimal amount of series-connected boreholes in the BTES was lower than expected. In most optimal cases in the 100, 200 and 500 building scenarios, only 1 to 3 boreholes were connected in series, while in the Drake Landing Solar Community it was 6 (Sibbitt et al., 2011). However, in the 50 building case typical values were between 4 and 9. The DLSC community consists of 52 houses, so the results are in line with practical implementation. It seems that having many boreholes connected in series is only important for relatively small communities.

As flowrate within each borehole loop was kept constant, having less seriality increased the amount of separate loops and thus the total flowrate through the BTES. The higher total flow increased both the general performance and the fraction of heat demand met directly by the BTES without the heat pump. Simply increasing the flowrate per pipe circuit should have a similar effect. A constant flowrate was used for discharging the BTES, but it might be beneficial to adjust the flowrate according to the current BTES and tank temperatures.

#### 395 4.1. Comparison to other studies

The achieved renewable energy fractions in this Finnish simulation study were very favorable com-396 pared to some actual German solar communities, as shown in Table 6. Even the low performance cases 397 exceeded in REF<sub>heat</sub> (solar fraction) both the Neckarsulm and Crailsheim communities. The main differ-398 ence in this study to the Neckarsulm community is the heat pump. With the help of a heat pump, high 399 renewable energy fractions can be achieved, even without solar thermal capacity. However, the Crailsheim 400 community also contained a heat pump. The difference between the better performing cases (100C and 401 500C) and Crailsheim can partly be attributed to the higher solar thermal area compared to heat demand, 402 but it is not the sole reason. The solar capacity to demand ratio was double for 500C compared to Crail-403 sheim, but of the same scale for case 500A. The BTES volume vs demand ratio was 5 times higher for 500C than for Crailsheim. However, even in the lower performance case 500A the REF was significantly higher for the simulated case compared to Crailsheim. Configurations in this study were optimized, so the results 406 can reasonably be expected to be better than those from non-optimal studies. However, no cost comparison 407 vs. Neckarsulm and Crailsheim was done and it is not known what the ratio of cost vs. performance is. Re-408 gardless, simulated studies are sure to be overly optimistic as real systems always suffer from unexpected 409 problems, such as component failures, less than ideal efficiencies, uncontrolled ground water flows and 410 different than expected environmental conditions. Such technical problems and non-ideal behaviour have 411 been studied in (Rehman et al., 2018). 412

	100A	100C	500A	500C	Neckarsulm (BTES+boiler)	Crailsheim (BTES+HP)
Heat demand (MWh/a)	709	723	3 554	3 615	1 891	4 100
ST area (m <sup>2</sup> )	531	2 637	5 569	14 372	5 007	7 300
BTES volume (m <sup>3</sup> )	140 078	24 702	276 588	179 497	63 360	37 500
ST/Heat (m <sup>2</sup> /MWh)	0.75	3.65	1.57	3.98	2.65	1.78
BTES/Heat (m <sup>3</sup> /MWh)	197.4	34.2	77.8	49.7	33.5	9.1
REF <sub>heat</sub>	65	88	70	92	39	50

Table 6: Comparison to other studies (Nussbicker et al., 2004; Schmidt and Mangold, 2006).

Another relevant case for comparison is the Drake Landing Solar Community in Canada, which reports

the measured performance of their energy system every month (Drake Landing Solar Community, 2012).

On year 2012, the DLSC achieved a high solar fraction, which is why the year was chosen for comparison 415 with the Finnish case in this study. From this study, case 100C (year 3) was chosen, due to similarity 416 to the DLSC case. Figure 24 shows the heating energy used in the district loop, relative to heated floor 417 area. The simulated Finnish case had higher total and summertime values because it included both space 418 heating and domestic hot water demand, while the DLSC case only included space heating. The simulated 419 system utilized very high efficiency buildings, which additionally used low temperature space heating (35 420 to 40  $^{\circ}$ C), while DLSC used higher temperatures (37 to 55  $^{\circ}$ C). This can partly explain why the demand in 421 the more northern Finnish location was less than in DLSC. Of course, annual weather differences will also 422

<sup>423</sup> significantly affect the performance.



Figure 24: Heating demand relative to heated floor area in the simulated Finnish case 100C ( $10\ 000\ m^2$  heated area) and the real Drake Landing Solar Community ( $11\ 600\ m^2$  heated area) in Canada. Finnish demand includes both DHW and SH, while DLSC includes only SH.

Figure 25 shows the incident and collected solar energy for both the simulated and real system. The Finnish case had a lower solar insolation and collection rate, as well as a lower solar thermal efficiency (26% vs 33%). A major difference between the locations is the winter solar potential. While the DLSC produced useful amounts of solar energy even in winter, solar generation in the Finnish case was reduced to almost zero during the November-January period. Collected solar energy vs. collector area was 42% lower in the Finnish case.

Figure 26 shows the energy used to charge the seasonal storage in the simulated (24700 m<sup>2</sup> BTES volume) and real (34000 m<sup>3</sup> BTES volume) systems. The charge profiles closely followed the solar profiles from Figure 25 in both cases. The BTES efficiency was 56% in the Finnish case and 50% in the DLSC. More energy relative to volume was injected and discharged in the Finnish case, but in absolute terms the values were similar. The Finnish case had significantly higher discharge rates during the January-March period, due to lower active solar input compared to the DLSC.

# 436 5. Conclusions

Multiobjective optimization of a solar district heating system with seasonal thermal energy storage was
done for four different community sizes. Based on the Pareto optimal results, large solar communities with
200 and 500 houses could reach lower LCC per floor area than the small communities of 50 and 100 houses,
but actual performance was not significantly different in the four size categories. The range for the annual
purchased electricity demand with respect to heated floor area was 30 to 54 kWh/m<sup>2</sup> for 50 buildings, 30 to
54 kWh/m<sup>2</sup> for 100 buildings, 28 to 46 kWh/m<sup>2</sup> for 200 buildings and 28 to 54 kWh/m<sup>2</sup> for 500 buildings.



Figure 25: Incident and collected solar energy for the simulated case 100C (2640  $m^2$  collector area) and the real DLSC (2293  $m^2$  collector area).



Figure 26: Energy transfer to and from the BTES in the simulated case 100C (24700  $m^3$  BTES volume) and the real DLSC (34000  $m^3$  BTES volume).

The life cycle cost ranges for these same cases were 477 to  $259 \in /m^2$  for 50 buildings, 420 to  $252 \in /m$ 

100 buildings, 456 to 273 €/m<sup>2</sup> for 200 buildings and 453 to 260 €/m<sup>2</sup> for 500 buildings, over a period of 25
 years.

The high temperature requirements of DHW demand made the inclusion of a heat pump a necessity for all community sizes. However, a clear benefit from larger community size was related to BTES performance. In the 200 and 500 building communities the BTES could provide as much as 44% of the required heating directly without utilizing the heat pump, while the 50 and 100 building communities were limited to less than 20%. Electricity consumption of heating systems was reduced by 80% when comparing the

<sup>451</sup> best performing optimal cases with those of the lowest cost.

Lowering heating demand through well insulated buildings with heat recovery systems was a high priority, as it was simpler and cheaper than adding more generation and storage capacity. Apartment buildings generally have more favorable ratio of external heat transfer area to living space than detached houses. If enough space is available for solar thermal installation, it might offer a solution to further reduce energy demand to obtain high renewable energy fractions more cost-effectively.

High seriality in the borehole connections of the BTES was important for the smallest community of 50
buildings, in which 4 to 9 boreholes were connected in series in the optimal configurations. In other sizes
it was less beneficial, as often only 3 or less boreholes were connected in series. Further study on the effect
of flowrate on optimal borehole connectivity is required.

This study has shown that solar thermal energy can be used to provide a significant fraction of winter heating even in high latitude Nordic countries. Larger systems reduce unit costs while increasing the

<sup>463</sup> potential of seasonal storage. However, more realized projects are needed to generate practical experience
 <sup>464</sup> on system design and operation and to lower costs through increased market activity.

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